

Static Balancing of Small Composite Turbine Rotors

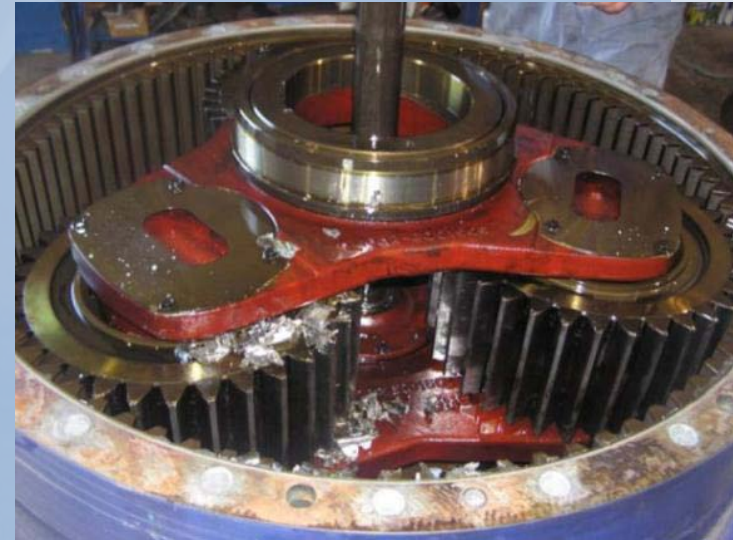


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Wind Turbine Rotor Balancing

- Current balancing technique
 - Pitch offset
 - Actively vary pitch to offset bending moments on rotor
 - Can reduce energy capture
 - Little information on proprietary methods in literature
- Why is balancing important?
 - Excessive noise
 - Safety in populated areas
 - Improved reliability and return on investment (ROI)
 - Could lead to gearbox fatigue failures



Need for Balancing: Comparison

- Car Wheel
 - 700 RPM on the freeway (100 km/h)
- 5g is smallest typical balance mass
 - Assume accuracy of wheel balance is $\pm 2.5g$



- Shake force is 1.5% of wheel weight
 - This is deemed acceptable

Need For Balancing Continued

- Vestas V90

- Shake force per blade up to 11.5 kN of due to construction tolerance alone
 - Approx. 3% of rotor weight
- Uneven Aerodynamic loads
- Wind gusts



- Bergey Excel 10

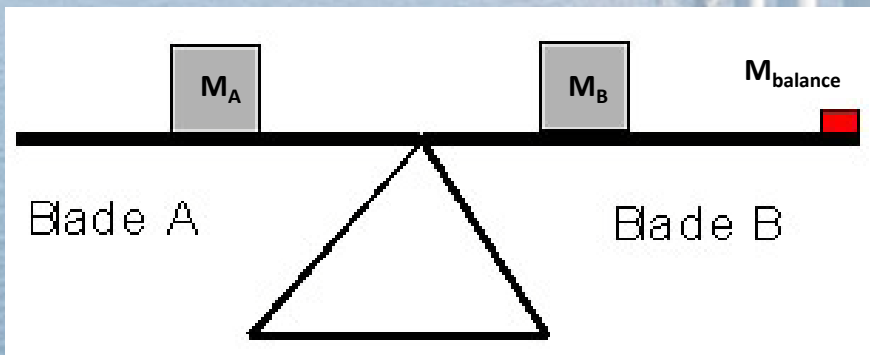
- A 1% blade weight tolerance at 400 RPM could cause an 360 N shake force
- 60% of rotor weight



Static and Dynamic Balancing

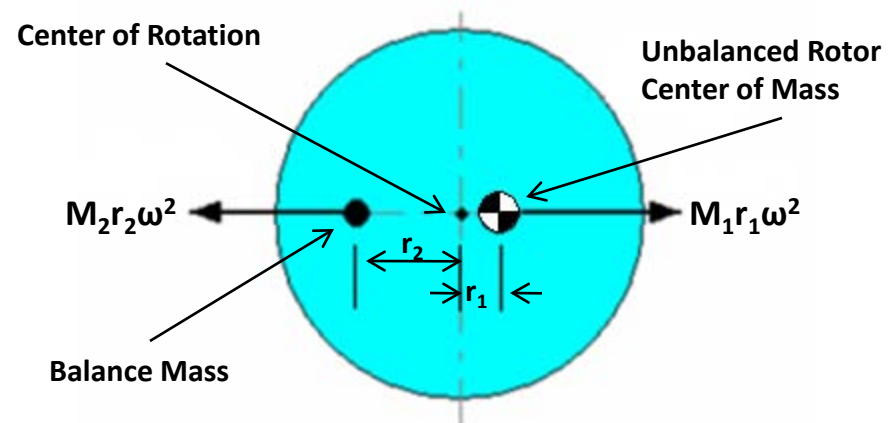
- Static

- Matching $R \times M$ of each rotating part while stationary
 - Safe
 - Not as accurate
 - Relatively economical in terms of space and cost



- Dynamic

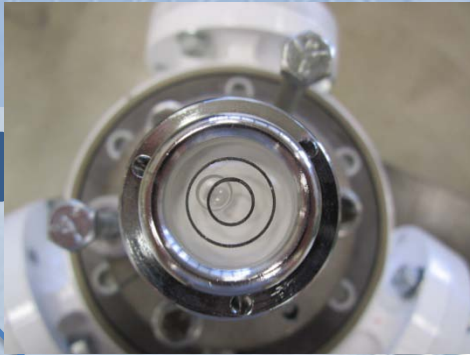
- Matching $R \times M \times \omega^2$ by rotating the part with a variable speed motor
 - Very accurate
 - Expensive
 - Requires more space
 - Impractical for wind turbine rotor



Static Balancing

Method 1: Full rotor

- Match mass moments of all blades and hub simultaneously

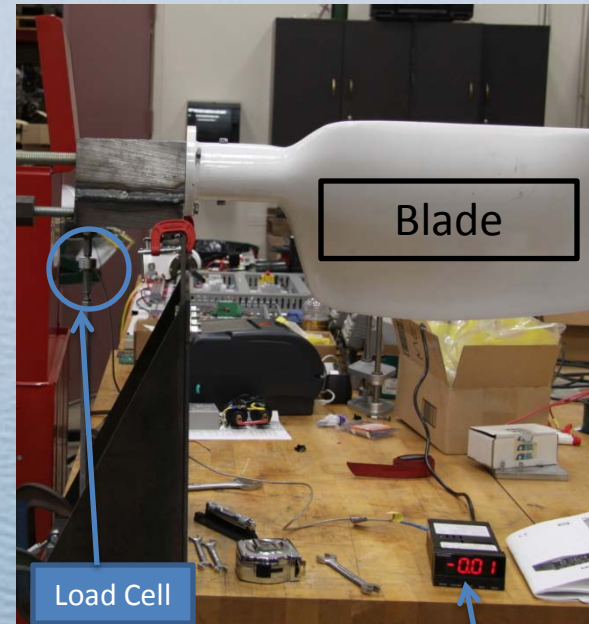


Top View



Method 2: Single blade

- Match mass moments of all blades to the same value sequentially



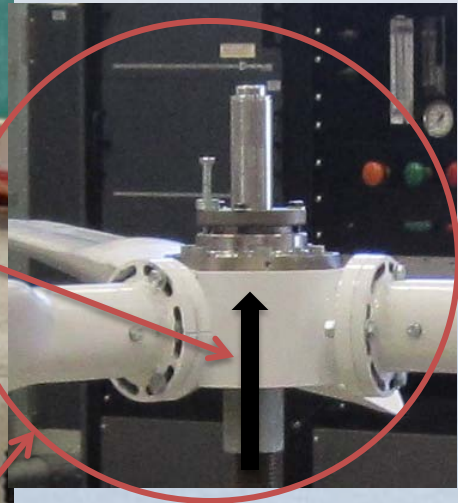
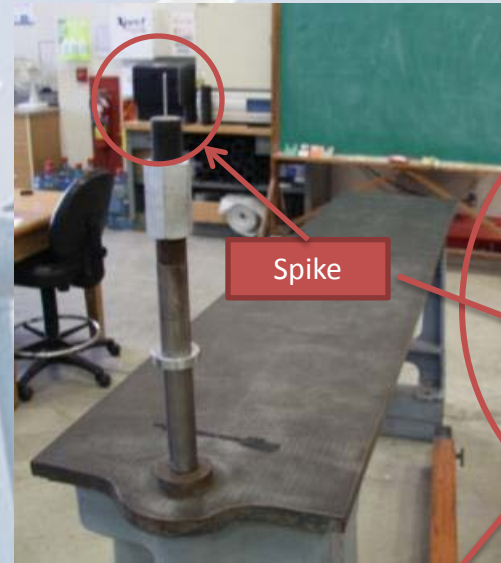
Blade

Load Cell

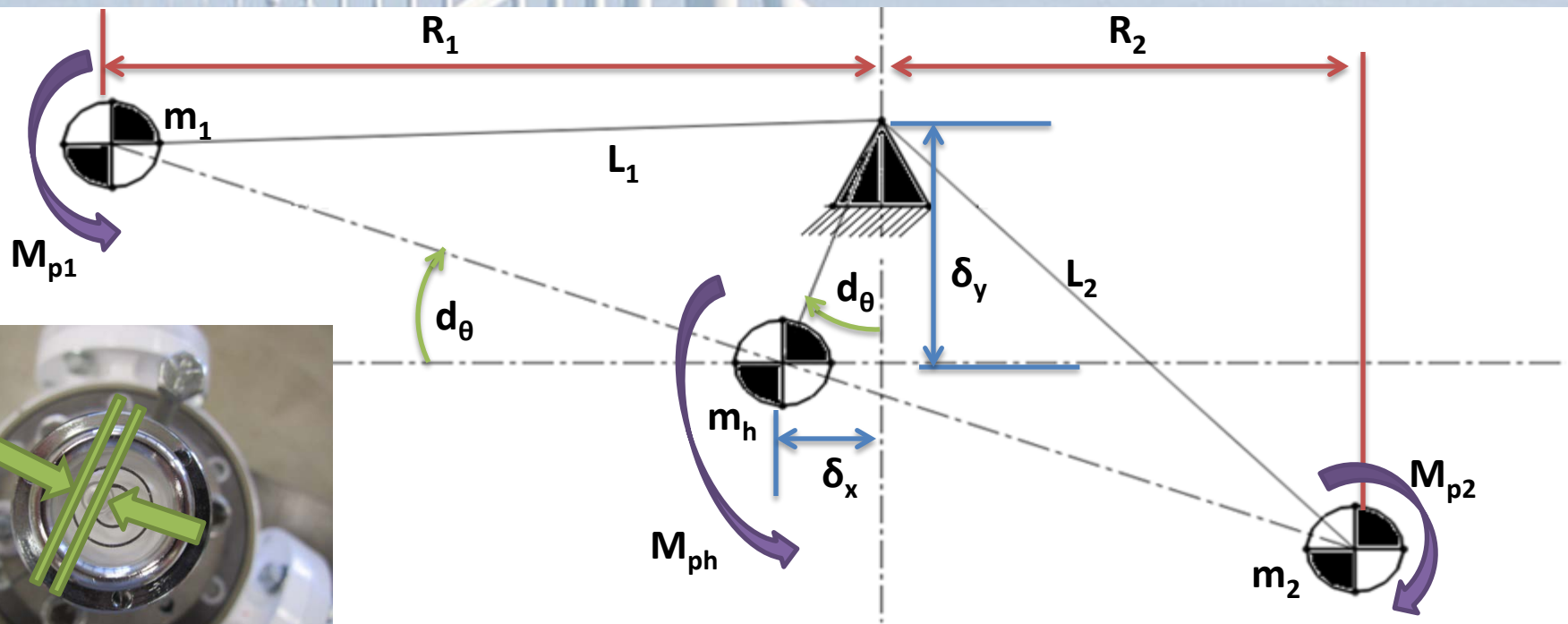
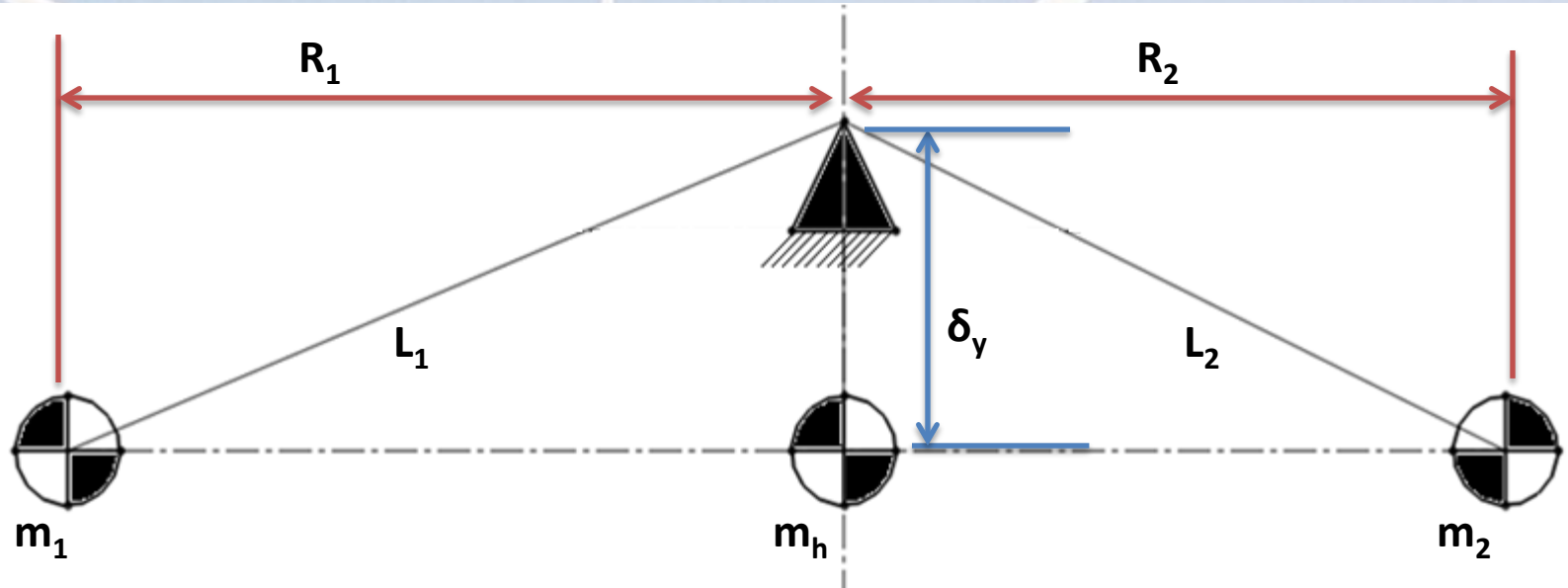
Load Cell Readout

Method 1: Full Rotor

- Full rotor is balanced on a spike or “frictionless” point
- Very sensitive
- Requires space the size of rotor swept area

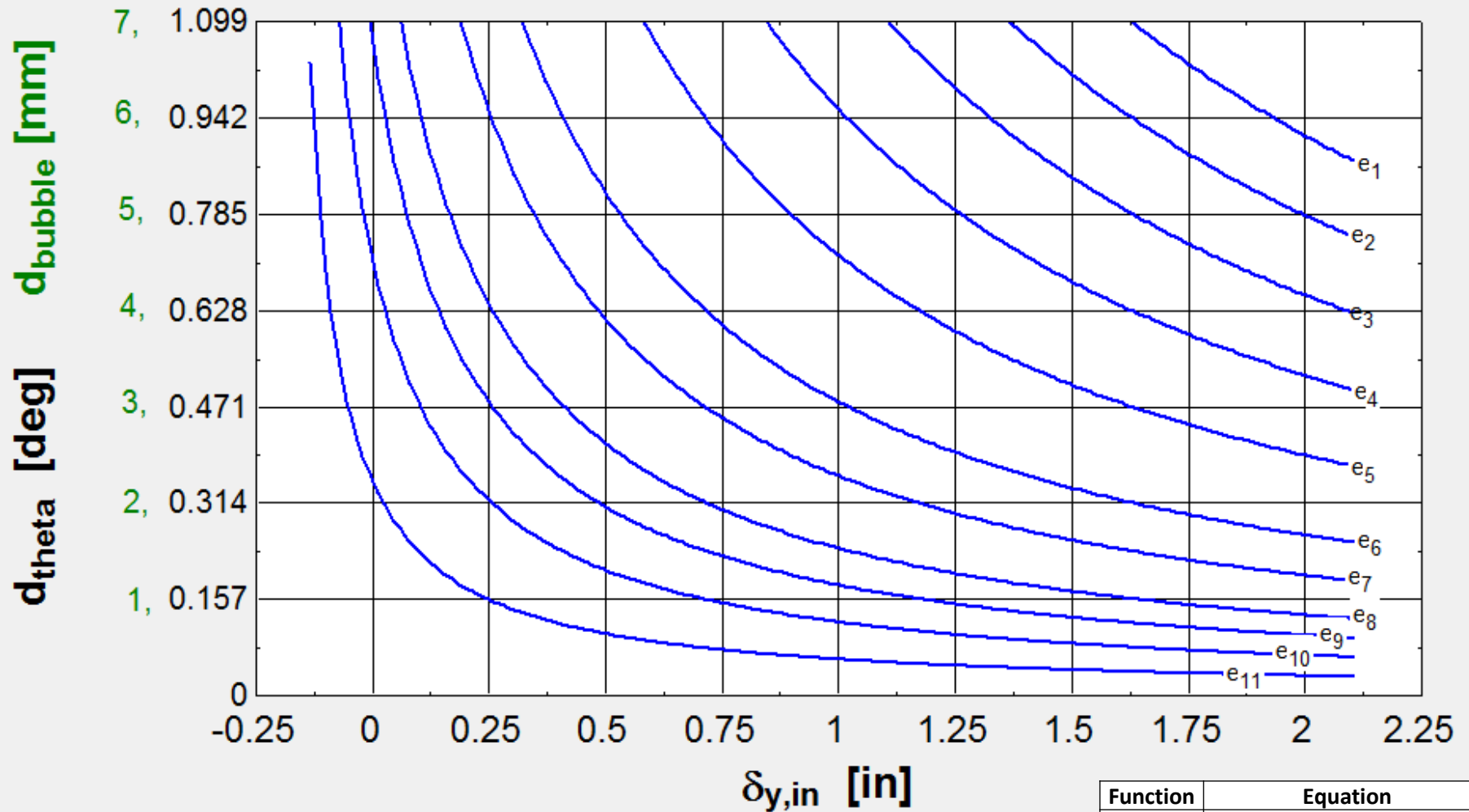


Full Rotor Method Formulation



Analysis of Full Rotor Error and Sensitivity

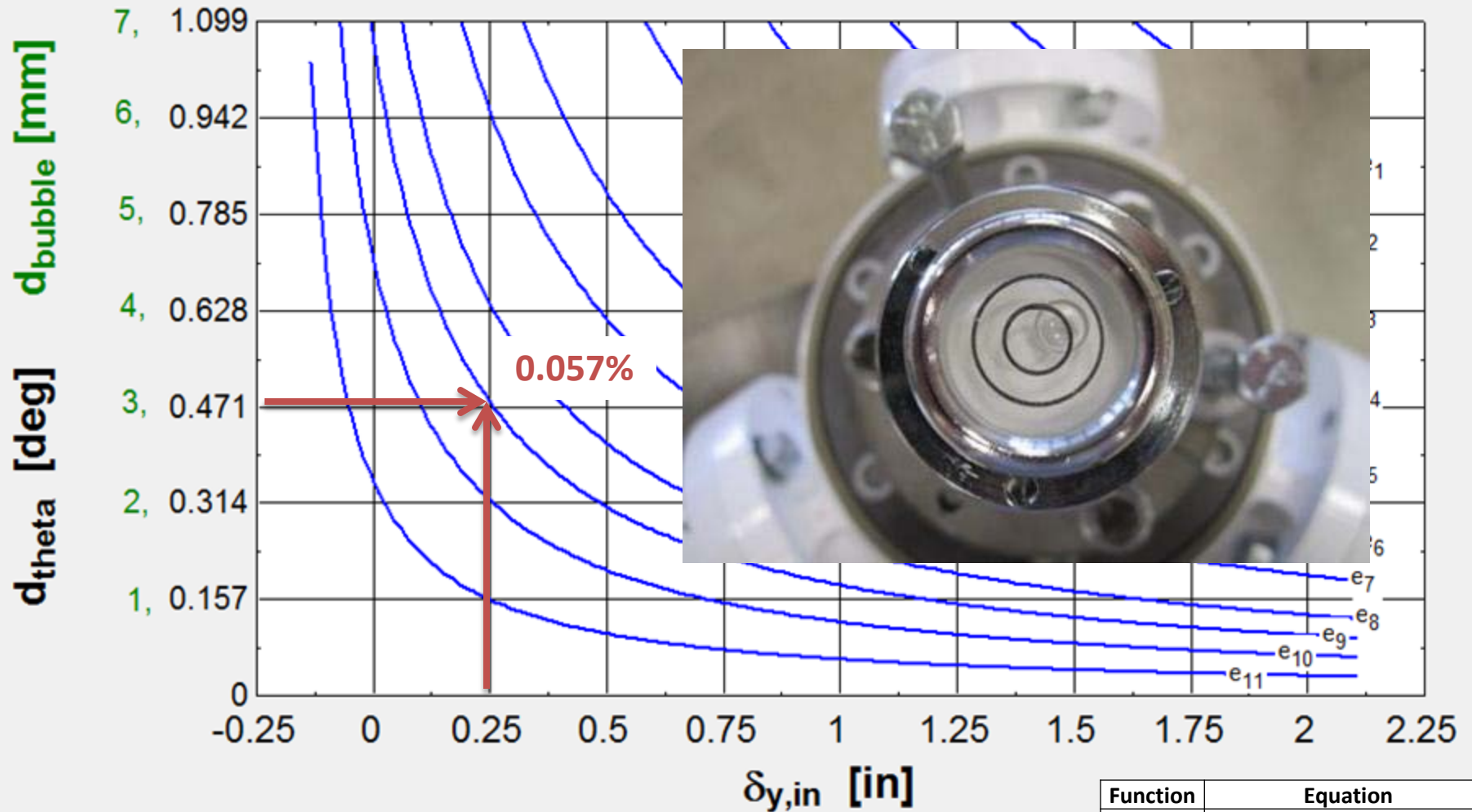
Rotor Tilt for Different Values of $R_2m_2/(R_1m_1+R_hm_h)$



Function	Equation
e_1	$B = 0.000299 \cdot d_{\theta} + 1.00408$
e_2	$B = 0.000298 \cdot d_{\theta} + 1.00350$
e_3	$B = 0.000297 \cdot d_{\theta} + 1.00292$

Analysis of Full Rotor Error and Sensitivity

Rotor Tilt for Different Values of $R_2m_2/(R_1m_1+R_hm_h)$



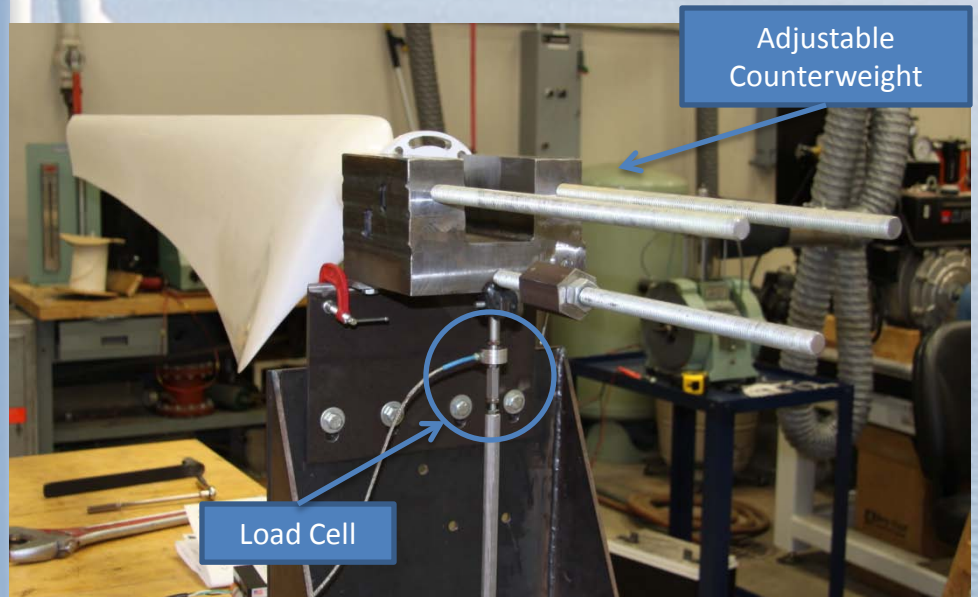
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Full Rotor Conclusions

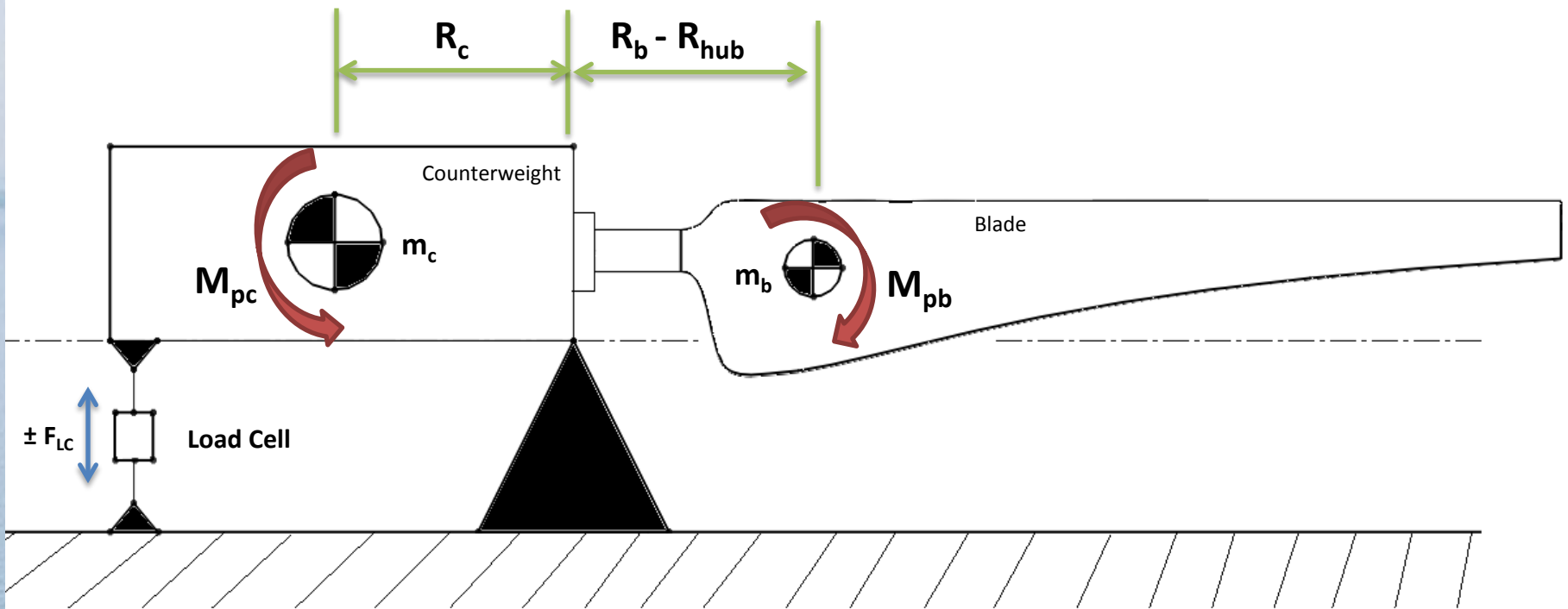
- Hub must be perfectly balanced independently
- Shows qualitative balance more than true quantified imbalance
- Actual balance is close to 0.5%
- Corresponds to a shake force
2.6 % of total rotor weight at design speed

Method 2: Single Blade

- Individual blades resting on a knife edge
- Must account for not having a hub, i.e. distance from C.G. to pivot plane is R_{hub} shorter than on full rotor method



Single Blade Method

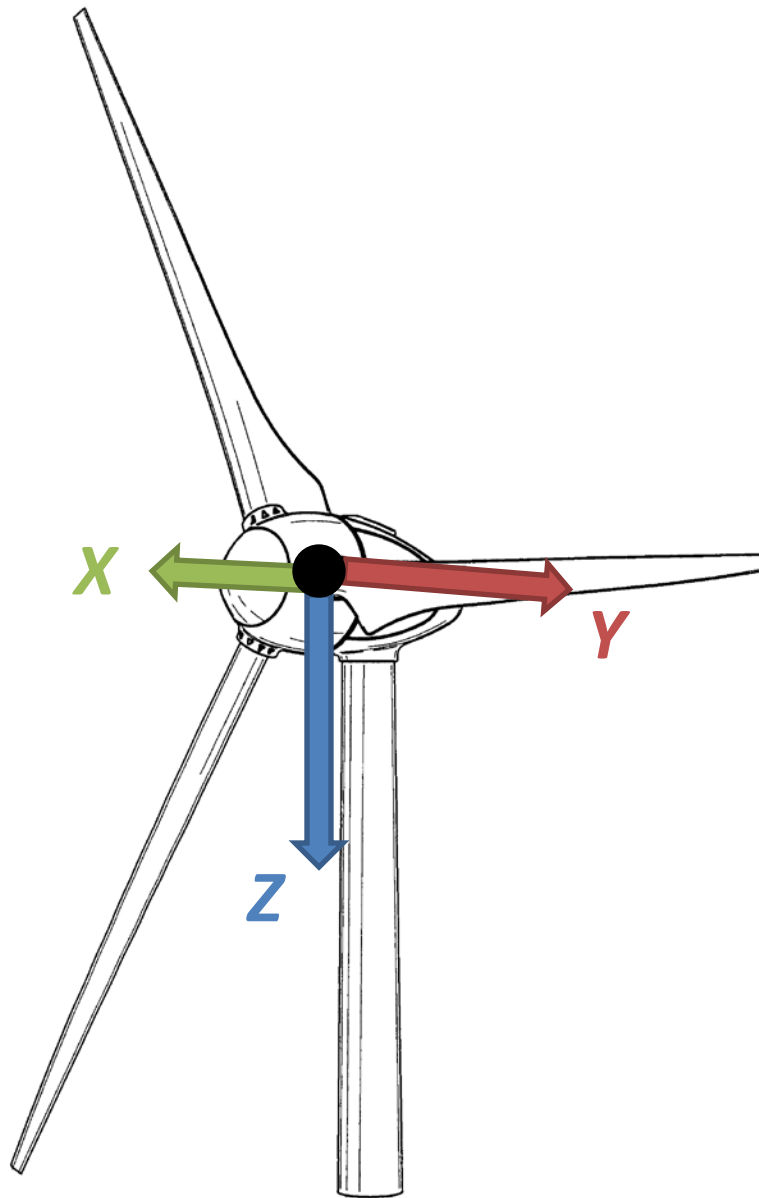


Errors and Result

- Load cell tolerance is critical
- Inaccuracy on pivot plane
 - Alignment
 - Rattle
 - Level
- Rotor within 1% balance
- Corresponds to a shake force 4% of total rotor weight at design speed
- With further refinement could be appropriate for utility scale

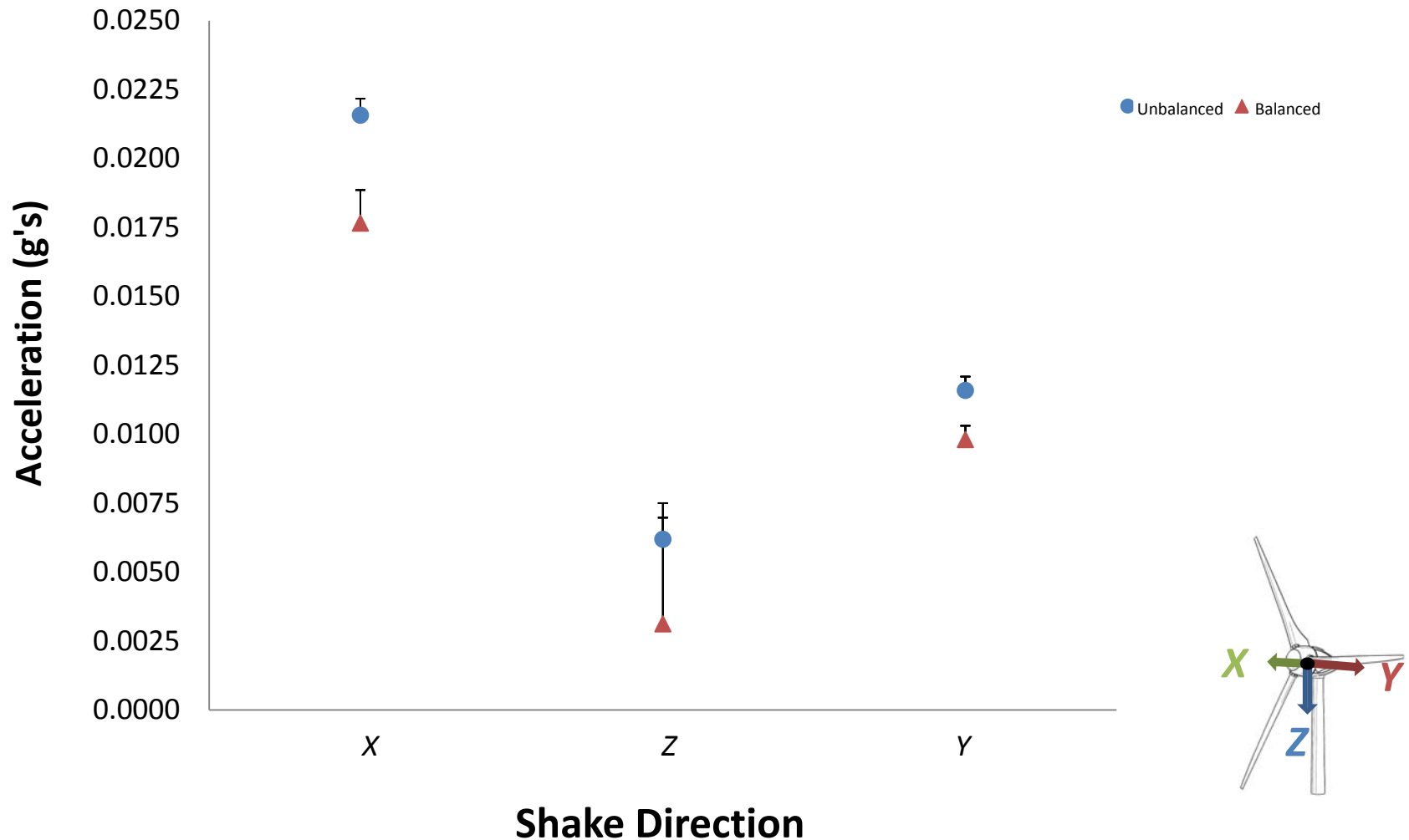


Experimental Verification



Experimental Verification

Effect of Rotor Balancing at Design Speed (208 RPM)



Thank You

- Bently Center for Engineering Innovation at Cal Poly
- California Central Coast Research Partnership (C³RP)



References

- Databases

- Engineering Village 2
- IEEE/IEE Electronic Library
- INSPEC
- Web of Science

- Websites

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